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(54) Heat Exchange Apparatus

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"HEAT EXCHANGE APPARATUS"

ABSTRACT OF THE DISCLOSURE

5 A heat exchange apparatus comprises a plastics
outer casing for mounting in the exterior wall of a build-
ing such that it extends outwardly of the building wall to
define an inlet and an outlet space from the wall. A
heat exchange block is mounted within the outer casing
and defines two paths of air therethrough mutually at right
angles and both inclined to the horizontal. The block is
10 formed of a plurality of polypropylene extruded sheets
(preferably Coroplast (trade mark)) which comprises two
thin sheets supported by a plurality of cells passing
along the sheets. The Coroplast sheets are separated by
spaces to form elongate air channels. The block is sup-
15 ported within the outer casing on resilient mountings and
is arranged such that the air exhausted from the building
passes through the elongate channels. A fan drawing air
into the building from the exterior through the heat ex-
change block includes a nozzle for directing the air into
20 the building away from the outlet. The fan can be revers-
ed periodically so that both fans act to draw warm air out
from the building through the block to melt any frost
forming within the block.

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HEAT EXCHANGE APPARATUS

BACKGROUND OF THE INVENTION

5 This invention relates to heat exchange apparatus and particularly such apparatus useful for ventilation air exchange in buildings.

10 It is well known to provide heat exchangers which allow heated air to be drawn from a room or building for exhaust to the exterior and to pass the air through a heat exchange block which defines a second path for incoming air whereby the outgoing air warms the incoming air to avoid undue heat loss from the building. The heat exchange block provides a plurality of layers or sheets of material which act to separate the two flows of air which provide the necessary heat exchange.

15 A number of problems exist with this type of unit, particularly when used for ventilation for a building used in livestock confinement. In fall and in winter, such a building is characterized by moderate temperature (60°F to 70°F) and high humidity. The temperature is
20 achieved and maintained both biologically (through metabolism of feed) and by mechanical heat sources. The high humidity is the direct result of animal metabolism, density of confinement and evaporation of wastes. The humidity is



highly corrosive in some types of livestock confinement operations and oxygen consumption in such environments is very high. Therefore, the primary environment management problem in livestock confinement operations has been meeting the high required rates of ventilation for oxygen replenishment and de-humidification while not giving up the moderate temperature conditions and while minimizing the cost of fuels necessary to maintain the temperature conditions.

Traditionally, confinement livestock operators have had two responses to these thermally contradictory circumstances, either they shut down for the coldest part of the year or they ventilate and heat at the same time at a great cost. There has been therefore a substantial need for a cheap, simple and efficient air-to-air heat exchanger to overcome this problem. The heat exchanger, operating across a temperature spread of as much as 100°F and very high humidity differentials must achieve efficiencies of heat transfer as much as 60% to 90%. At lower efficiencies, significant temperature drops in the barn may be experienced. At these higher efficiencies, the amount of heat recovered from the outgoing air is sufficient that it tends to cause freezing in the exhaust part of the exchanger. There has been a need therefore to include a

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defrosting mechanism which enables the use of the higher efficiencies which are necessary to maintain the temperature within the barn.

5 A further problem particularly relevant to the above conditions is the very high corrosiveness of the exhausted gases. This corrosiveness can have very serious effects upon any metal parts within the heat exchanger.

10 A yet further problem is that the incoming air must be circulated or distributed within the room, particularly bearing in mind the high density of confinement and the high requirement for oxygen caused thereby. It is possible to provide such distribution of the air by additional duct work within the building but this substantially increases cost and complexity, bearing in mind
15 that many buildings may be of a structure not suitable to receive such duct work.

SUMMARY OF THE INVENTION

It is one object of the invention therefore to provide a heat exchange apparatus which is suitable for
20 use in the difficult area of livestock confinement buildings. According to the invention there is provided a heat exchange apparatus comprising an outside casing, means defining in the casing a first inlet, a first outlet, a second inlet and a second outlet, means for mounting the
25 outside casing relative to an outside wall of a building such that the first inlet and the second outlet are arrang-

ed on one side of the wall facing within the building and the first outlet and the second inlet are arranged on the other side of the wall facing outside the building, a heat exchange block defining therethrough two separate air paths mutually at right angles, means mounting the block in the casing such that air can pass in a first and a second separate path respectively from said first inlet through said block to said first outlet and from said second inlet through said block to said second outlet, a first fan for pumping air in said first path and a second fan for pumping air in said second path, said heat exchange block comprising a plurality of thin sheets of plastics material lying in planes parallel to the first and second flows and arranged so as to divide the first and second flows into a plurality of alternate portions separated by the sheets, a first plurality of elongated spacer strips arranged to support and space each sheet on one side thereof relative to the adjacent sheet on that side, the strips being parallel to the second air path as it passes through the block and spaced by a distance of the order of the spacing between the sheet and the adjacent sheet and a second plurality of elongated spacer strips on the other side of each sheet and arranged parallel to the first air path as it passes through the block and spaced so as to support said sheet at positions spaced widely relative to said spacing between the sheet and the next adjacent sheet.

It is one advantage of the invention that the heat exchange block is formed of commercially available plastics sheet provided as a polypropylene extrusion which can be separated by spacer strips so that the path for the outgoing air is substantially free of spacer strips to avoid contamination while the individual sheets which separate the two paths of air can be very thin thus providing good heat exchange contact.

It is a still further advantage of the invention that the heat exchange block can be readily mounted within a plastics outer casing by resilient mounting means thus avoiding damage to the heat exchange block from vibration caused by the fans.

With the foregoing in view, and other advantages as will become apparent to those skilled in the art to which this invention relates as this specification proceeds, the invention is herein described by reference to the accompanying drawings forming a part hereof, which includes a description of the best mode known to the applicants and of the preferred typical embodiment of the principles of the present invention, in which:

DESCRIPTION OF THE DRAWINGS

Figure 1 is a cross sectional view of a heat exchange apparatus according to the invention.

Figure 2 is a cross sectional view along the

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line 2-2 of the apparatus of Figure 1.

Figure 3 is a front elevational view of the apparatus of Figure 1.

5 Figure 4 is an enlarged end view of the embodiment of Figure 1 showing the interior of the heat exchange block.

Figure 5 is an enlarged end view of the embodiment of Figure 1 showing the heat exchange block from the other direction.

10 In the drawings like characters of reference indicate corresponding parts in the different figures.

DETAILED DESCRIPTION

15 The heat exchange apparatus is shown in detail in Figures 1, 2 and 3 mounted in a wall 10 comprising the exterior wall of a livestock confining building. The heat exchange apparatus is generally indicated at 20 and comprises an outside casing 21 providing a generally rectangular main body having upper and lower walls 22, 23, an outer depending wall 24 and sides 25, 26, only the former of which is visible in Figure 1. An open front portion of the main body of the outside casing 21 includes a surrounding flange 27 for coupling with a front plate 28 by suitable pop rivets 29 as shown in Figure 3.

20 The flange 27 and the outer portion of the front face 28 provide a supporting flange whereby the heat
25

exchange apparatus can be secured to the wall. Specifically, an opening is provided in the exterior wall of the building of the correct size to receive the body of the heat exchange apparatus with the flange abutting the face of the wall around the outside of the opening. The heat exchanger is then secured to the wall by screws passing through openings 30 provided through the flange 27 and the front face 28 and passing into the wall. In this way, the front face 28 is substantially flush with the wall and the main body of the heat exchanger extends out through the wall to avoid taking up space inside the building.

The main body 21 comprises rotational moulded polyethylene and thus is formed as a single piece without seams. The upper wall 22 is inclined slightly from the horizontal and lies parallel to the lower wall 23 which is similarly inclined so that any liquid within the outer casing either drawn in by the air or condensed from the air can leave the casing by running along the lower wall 23 to an outlet nozzle 32 or to a drain hole 31. The outlet nozzle 32 comprises a slot extending across the full width of the heat exchanger and defined at the junction between the wall 23 and the end wall 24 which extends downwardly and outwardly from the upper wall 22 towards the lower wall 23. A second air inlet 33 is provided in

each of the side walls 25, 26 of the main body 21 as a circular opening adjacent the walls 22, 24. Thus, the second inlet 33 is arranged at right angles to the first outlet 32.

5 Each of the walls 22, 23, 24 includes a notch
(not shown) extending across its width on the interior
thereof. As the wall thickness is constant, this shows a
small ridge on the outside of the heat exchanger. The
notches, 35, if provided, are arranged at suitable posi-
10 tion to receive the three edges of a rectangular heat ex-
change block 36. Each of the three edges includes a strip
37 of foam rubber which is secured to the edge of the
block 36 by adhesive and is located by the notch 35. As
shown in Figure 2, the width of the heat exchange block 36
15 is the same as that of the outside casing and hence the
sides of the block lie closely adjacent or in contact
with the sides 25, 26 of the main body 21. The fourth
edge of the block 36 shown at 38 is supported relative to
the main body 21 by a plastics strip 39 which is secured
20 at its ends to the sides 25, 26 by suitable screws. Thus,
it will be appreciated that in manufacture with the strip
39 not yet in place, the block 36 can be readily inserted
into the main body 21 to a position where the edges carry-
ing the foam strips 37 engage the notches 35 following
25 which strip 39 is positioned to locate the block 36 in

13

place. The block 36 is thus mounted within the main body 21 in a resilient manner which avoids the communication of any vibration of the main body 21 to the heat exchange block 36.

5 Turning now to Figures 4 and 5, the heat ex-
change block 36 comprises a large number of sheets of
Coroplast (registered trade mark) arranged in parallel re-
lation. Coroplast is a commercially available polypropy-
lene extrusion comprising two parallel sheets separated
10 by a plurality of parallel spacer strips running along the
length of the sheets. The spacing between the strips is
of the order of the spacing between the sheets so as to
form a large number of square cells across the sheet. The
edge of each sheet of Coroplast is shown in Figure 5 where
15 the cells are readily visible and are shown at 40.

In Figure 4, the side of each of the sheets can
be seen at 41 and it will be appreciated that the portion
visible in Figure 4 is that of the side of the endmost
spacer strip together with the edges of the individual
20 sheets making up the Coroplast sheet.

Each of the Coroplast sheets is separated from
the next adjacent Coroplast sheet by a plurality of
spacers of which there are shown in Figure 4. Each spacer
indicated at 42 comprises a narrow strip of Coroplast con-
25 sisting of two or three cells defined by short widths of

the sheet attached by two or three of the spacers.

In manufacture, the polypropylene extrusion comes in large sheets which are firstly cut to size. Subsequently, the heat exchange blocks are built up by glue-
5 ing to a first cut sheet, three spacer strips 42. Next a second sheet of the extrusion is glued upon the spacer strips and in a similar manner, the whole heat exchange block is built up. In one example, there are thirty-eight sheets of the polypropylene extrusion with a similar num-
10 ber of separating channels defined by the spacers 42. The number of spacers 42 used in each channel is dependent upon the size of the sheets and is sufficient merely to provide adequate support of the sheets to define the channels therebetween. It may in fact only be necessary
15 to provide spacers at either end of the sheets.

It will be appreciated therefore that the heat exchange block provides two separate air paths through the block mutually at right angles. The first path can be seen from Figure 5 where the air can pass through the
20 polypropylene extrusions and specifically through each of the cells defined thereby. The second path is shown in Figure 4 and is formed by the channels between each of the extrusions and defined by the spacers 42. Thus, the air paths are separated into a plurality of alternate
25 layers separated only by the thin surface sheets of the

extrusions. As the thin surface sheets are supported closely by the spacer strips shown in Figure 5, the surface sheets themselves can be very thin since they need little strength. However, in the other direction shown in Figure 4, every few spacers are needed since each extrusion is itself relatively strong in view of the large number of cells formed.

Referring again to Figure 1, it will be noted that the heat exchange block 36 acts to divide the main body 21 into four separate sections. The first section lies adjacent the outlet nozzle 32 and communicates with one side of the block 36. A second section lies adjacent the second inlet 33. The opposite end of the block 36 from the outlet 32 defines a third section of the main body 21 and similarly, the opposite end of the block 36 from the second inlet 33 defines the fourth section. The third section is separated from the fourth section by the strip 39.

The third section communicates with a circular opening 45 defined in the front face 28 and providing an inlet for air exiting from the interior of the building. Within the opening 45 is provided a fan 43 of conventional construction comprising a motor 431 and a supporting grid 432 by which it is secured to the front face 28. The fourth section is connected to a circular opening 46 also

provided in the front sheet 28 in which a fan 44 is similarly mounted including a motor 441 and a supporting grid 442.

5 A control device 47 supplied with electrical power from the building acts to drive the fan 43 in a direction to pump air from the interior of the building through the first inlet defined by the opening 45, through the heat exchange block 36 and out through the first outlet 32. It will be noted from Figure 4 that the path that
10 the air takes from the building to the exterior through the heat exchange block 36 passes through the channels defined between the extrusions and hence through long unobstructed passages with spaces provided only infrequently. Thus the channels are relatively open to resist the de-
15 posit of contaminants from the air and liquid condensing from the air. In addition, the relatively long channels are easy to clean by removal of the fan 43 following which water can be injected into the heat exchanger along the normal path of air to wash the path free from the contaminants which have collected.
20

The control device 47 acts to drive the fan 44 normally in a direction to draw air through the heat exchange block from the second inlet 33 and to direct it into the interior of the building. Since the air coming
25 from the outside of the building is relatively clean and

free from moisture, it is less prone to deposit liquid and contaminants within the heat exchange block and hence the small cells supporting the extrusions are less likely to become contaminated.

5 A nozzle device 48 and shown in Figures 1 and 3
is mounted over the fan 44 and is secured to the front
face 28 by suitable means now shown. The nozzle comprises
sides 481, 482 and upper and lower walls 483, 484. The
upper and lower walls include portions which converge to-
10 ward a narrow elongate slot 485 defining a nozzle through
which the whole of the air coming into the building passes.
Thus the air is accelerated to a high speed of the order
of 2000 per/min. In this way the air is directed away
from the outgoing air drawn by the fan 43 to avoid fresh
15 air being drawn back out by the fan 43 and in addition
the high speed of the air acts to distribute the air
through the building without the necessity of additional
duct work. It will also be noted that the nozzle 32 is
directed away from the inlet 33 so that stale contaminat-
20 ed air is now drawn in through the inlet 33. Particular-
ly, the inlet 33 is arranged at right angles to the nozzle
32 and also at right angles to the upper surface of the
main body 21 thus avoiding the ingress of rain or snow.
Furthermore, the majority of the heat exchanger lies out-
25 side the building whereby the nozzle 32 lies a consider-

able distance from the wall 10 of the building so that any liquid running from the nozzle falls to the ground well away from the building and away from the foundation.

After a period of time dependent upon the efficiency of the heat exchange block and the relative temperatures inside and outside the building, frost develops on the heat exchange block within the path of air exiting from the building and mainly, adjacent the outlet 32. If this frost is allowed to continue to develop and to accumulate the dust and contaminants carried by the air stream, it will eventually seal off the heat exchange block and prevent the further outflow of the stale air. When this occurs or as the condition is developing, the amount of heat introduced into the air drawn into the building gradually reduces. A temperature sensor 49 provided within the nozzle 48 acts to continually measure the temperature of the incoming air. When the temperature reaches a predetermined low as set upon and determined by the control device 47, the control device 47 acts to reverse the polarity of the motor 44 whereby the direction of the fan 44 is reversed to draw air from the interior of the building out through the heat exchange block to exit through the inlet 33. Thus both the fans 43 and 44 are acting during this period to draw air out of the building through the heat exchange block so that the warm air in

both air flow paths acts to rapidly melt any frost from the walls of the heat exchange block 36 whereupon the frost and melted liquid can drop from the block into the outside casing adjacent the nozzle 32. The control device 47 is arranged to control the reversal of the fan 44 for a predetermined time period following which the fan is reverted to its normal direction to continue normal operation of the heat exchange apparatus.

In an alternative arrangement, the temperature sensor 49 is omitted and the control device 47 acts to time both the period of normal running and the period of reverse. The former can be set to accommodate different conditions, but in practice a reverse period of 8 minutes in every hour has been found to be satisfactory.

In view of the fact that the frost forms in the unobstructed channels of the heat exchange block, the fact that the heat exchange block is inclined so that the flow of air is inclined to the horizontal, the fact that the plates of the heat exchange block are arranged in vertical planes, and the fact that both air channels are used to transport warm air, the developed frost is very rapidly ejected from the heat exchange block either in the form of liquid or as ice for later ejection through the nozzle 32.

As the whole apparatus excluding the fans and

their associated motors is manufactured of plastic material, problems of corrosion do not occur. The heat exchange block provides a very high efficiency in view of the very thin sheets separating the two paths of air. In addition the resilient mounting of the block within the outer casing avoids any damage to the delicate separating sheets caused by the continual vibration of the fans 43 and 44.

Since various modifications can be made in my invention as hereinabove described, and many apparently widely different embodiments of same made within the spirit and scope of the claims without departing from such spirit and scope, it is intended that all matter contained in the accompanying specification shall be interpreted as illustrative only and not in a limiting sense.

WHAT WE CLAIM AS OUR INVENTION:

(1) Heat exchange apparatus comprising an outside casing, means defining in the casing a first inlet, a first outlet, a second inlet and a second outlet, means
5 for mounting the outside casing relative to an outside wall of a building such that the first inlet and the second outlet are arranged on one side of the wall facing within the building and the first outlet and the second inlet are arranged on the other side of the wall facing
10 outside the building, a heat exchange block defining there-through two separate air paths mutually at right angles, means mounting the block in the casing such that air can pass in a first and a second separate path respectively from said first inlet through said block to said first
15 outlet and from said second inlet through said block to said second outlet, a first fan for pumping air in said first path and a second fan for pumping air in said second path, said heat exchange block comprising a plurality of thin sheets of plastics material lying in planes parallel
20 to the first and second flows and arranged so as to divide the first and second flows into a plurality of alternate portions separated by the sheets, a first plurality of elongated spacer strips arranged to support and space each sheet on one side thereof relative to the adjacent sheet
25 on that side, the strips being parallel to the second air path as it passes through the block and spaced by a dis-

tance of the order of the spacing between the sheet and the adjacent sheet and a second plurality of elongated spacer strips on the other side of each sheet and arranged parallel to the first air path as it passes through the block and spaced so as to support said sheet at positions spaced widely relative to said spacing between the sheet and the next adjacent sheet.

(2) Apparatus according to Claim 1 wherein the sheets are arranged in vertical planes.

(3) Apparatus according to Claim 1 each sheet its respective adjacent sheet and the first plurality of spacer strips being formed integrally from a polypropylene extrusion.

(4) Apparatus according to Claim 3 wherein the polypropylene extrusion is Coroplast (trade mark).

(5) Apparatus according to Claim 1, 2 or 3 wherein the first and second outlets each include nozzle arranged such that air emitted by said first and second outlets is deflected away from the second and first inlets respectively, the casing including a lower wall inclined to the horizontal and the nozzle means of the first outlet being arranged at the lower end of the lower wall to allow the exit of liquid from the casing.

(6) Apparatus according to Claim 5 wherein the nozzle means is arranged such that the speed of air issuing from said nozzle means of the second outlet is sufficient

to cause circulation of air within the building.

(7) Apparatus according to Claim 5 wherein the nozzle means is arranged such that the speed of the air is greater than.

5 (8) Apparatus according to Claim 5 wherein said second inlet is provided by a pair of openings each arranged in a respective side of the casing at right angles to said lower wall.

10 (9) Apparatus according to Claim 5 wherein said heat exchange block comprises a plurality of thin sheets of plastics material lying in planes parallel to the first and second flows and arranged so as to divide the first and second flows into a plurality of alternate portions separated by the sheets, a plurality of elongate spacer
15 strips arranged to support and space each sheet on one side thereof relative to the adjacent sheet on that side, the strips being parallel to the air flow on the respective side and spaced by a distance of the order of the spacing between the sheet and the adjacent sheet and a
20 second plurality of elongate spacer strips on the other side of each sheet and arranged parallel to the air flow on the respective side of the sheet and spaced so as to support said sheet only occasionally.

25 (10) Apparatus according to Claim 9 wherein the heat exchange block is arranged such that the first flow passes through the alternate portions defined on said other side of each sheet.

(11) Apparatus according to Claim 5 wherein said outside casing is formed from plastics material.

(12) Apparatus according to Claim 5 wherein said mounting means is arranged such that the outside casing extends through the outside wall of the building where-
5 by the first outlet and the second inlet are spaced from the outside wall of the building.

(13) Heat exchange apparatus comprising an outside casing, means defining in the casing a first inlet,
10 a first outlet, a second inlet and a second outlet, means for mounting the outside casing relative to an outside wall of a building such that the first inlet and the second outlet are arranged on one side of the wall facing within the building and the first outlet and the second
15 inlet are arranged on the other side of the wall facing outside the building, a heat exchange block defining there-through two separate air paths mutually at right angles, means mounting the block in the casing such that air can pass in a first and a second separate path respectively
20 from said first inlet through said block to said first outlet and from said second inlet through said block to said second outlet, a first fan for pumping air in said first path and a second fan for pumping air in said second path, said heat exchange block comprising a plurality of thin
25 sheets of plastics material lying in planes parallel to the first and second flows and arranged so as to divide

the first and second flows into a plurality of alternate portions separated by the sheets, a first plurality of elongated spacer strips arranged to support and space each sheet on one side thereof relative to the adjacent sheet
5 on that side, the strips being parallel to the second air path as it passes through the block and spaced by a distance of the order of the spacing between the sheet and the adjacent sheet and a second plurality of elongated
10 spacer strips on the other side of each sheet and arranged parallel to the first air path as it passes through the block and spaced so as to support said sheet at positions spaced widely relative to said spacing between the sheet and the next adjacent sheet, each sheet, its respective
15 adjacent sheet and the first plurality of spacer strips being formed integrally from a polypropylene extrusion.

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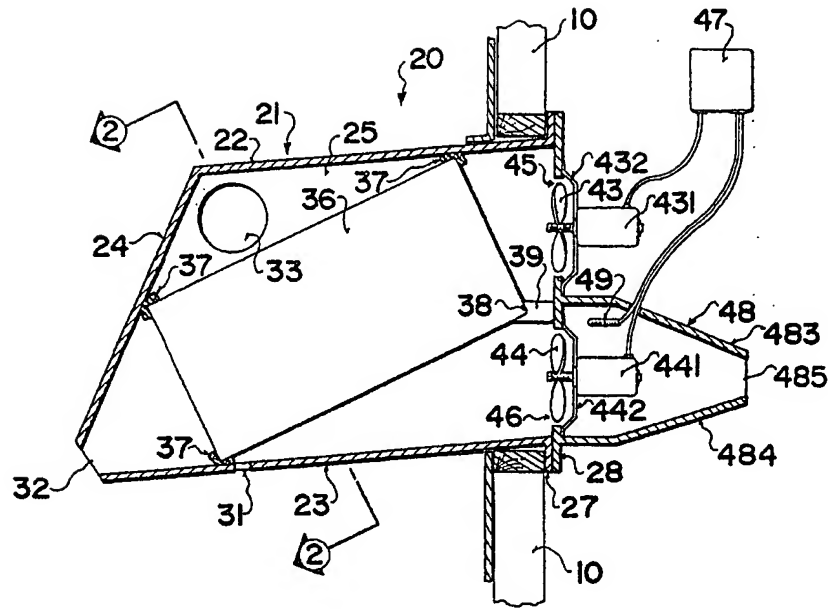


FIG. 1

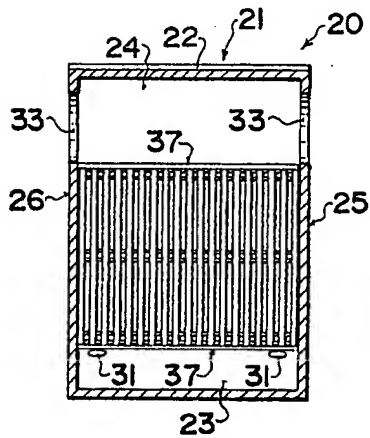


FIG. 2

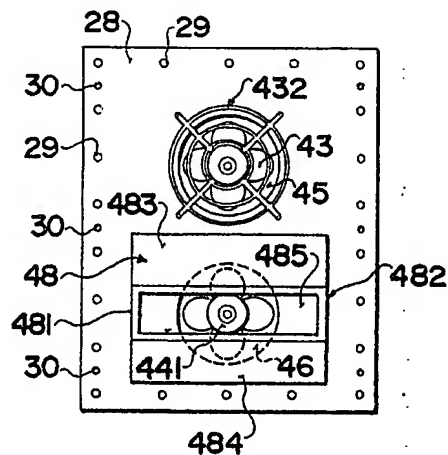
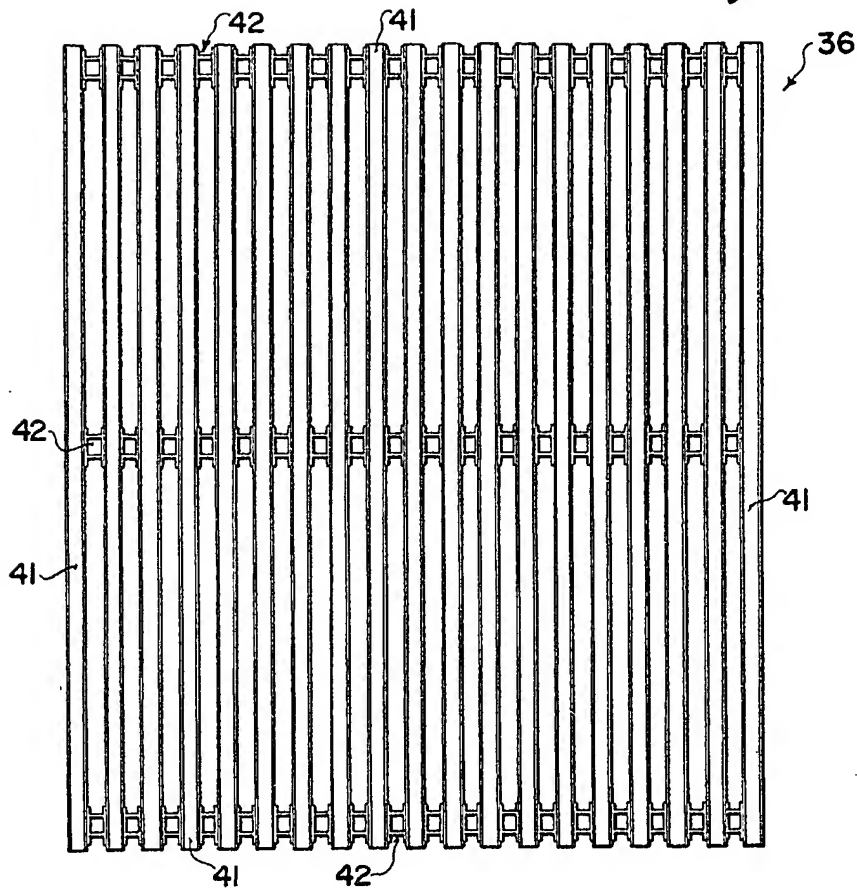


FIG. 3

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FIG. 4

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